

## **The Vapor Pressure of Indium, Silver, Gallium, Copper, Tin, and Gold Between 0.1 and 3.0 Bar**

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The vapor pressure of several liquid metals was measured using a method based on the gas-controlled heat pipe. Small samples of the test material were placed in a tungsten tube and heated to temperatures above 2900 K. The vapor pressure was measured using a gas-buffered pressure transducer and the vapor temperature was inferred from the tube surface temperature, which was measured with an optical pyrometer. Most of the tests were terminated by the failure of the containment tube. The measured pressures agree well with those calculated by thermodynamic methods from data at lower temperatures.

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**KEY WORDS:** copper; gallium; gold; heat pipe; high temperature; indium; liquid metals; silver; tin; vapor pressure.

### **1. INTRODUCTION**

Vapor pressure data for liquid metals at high temperature are sparse because accurate measurements are difficult to make. Available vapor pressure data, obtained by different investigators, may range over an order of magnitude at the same temperature. A measurement method based on the gas-controlled heat pipe has been developed relatively recently which offers improved accuracy over earlier methods. This new method, which may be described as a direct, dynamic measurement method using a gas-buffered pressure transducer, was first used by Bohdanský and co-workers at the Joint Research Centre at Ispra [1-4]. In the present paper, the

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extension of the heat-pipe method to temperatures in excess of 2900 K is described and new results are presented for the vapor pressures of In, Ag, Ga, Cu, Sn, and Au in the pressure range 0.1 to 3.0 bar.

## 2. METHOD

The heat-pipe apparatus used to measure vapor pressure is shown schematically in Fig. 1. A central, vertical, metal tube, the heat pipe, is lined on the inside with a capillary structure and filled with an inert gas at a certain pressure and with a small amount of the material to be investigated. The lower part of the heat pipe, called the evaporator, is heated and the upper part, the condenser, is cooled. Under steady-state operating conditions, a thin liquid layer, guided by the capillary structure, covers the wall of the heat pipe. Vapor, produced in the evaporator, flows up the center of the pipe to the condenser, where it is converted back into the liquid phase. The liquid returns to the evaporator along the wall of the tube, completing the test fluid circuit. The inert gas is swept to the top of

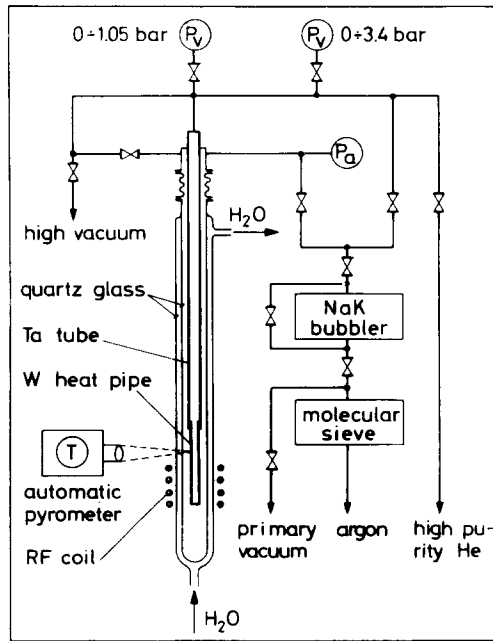


Fig. 1. Schematic experimental arrangement for vapor pressure measurements by the heat-pipe method at very high temperatures.

the condenser by the upward flowing vapor. This gas fills an extension tube which connects the heat pipe and the pressure transducer and blocks the flow of vapor. The heat transfer rate through this long gas plug and the surrounding thin-walled tube is low and the pressure transducer operates at room temperature.

The pressure drop in the vapor flow near the downstream end of the condenser is extremely small so that this section of the heat pipe operates under virtually isothermal conditions at the inner wall. The inert gas plug is in dynamic equilibrium with the vapor column beneath it. As long as the gas pressure is kept constant, the vapor pressure and, therefore, the temperature of the isothermal zone remain constant. A variation in the heat input causes a change in the length of the isothermal zone, increasing or decreasing the condenser area to maintain a heat balance, but does not influence the temperature. This provides a means for adjusting the length of the isothermal zone so that its temperature can be conveniently measured.

The heat pipes used for these tests were made from chemical vapor-deposited tungsten tubes. Each tube was 130 mm long, with a 12-mm outside diameter and an 8-mm inside diameter. The inner surface was lined with 30 axial grooves having a square cross section of  $0.4 \times 0.4$  mm. These grooves formed the capillary structure. Each pipe was plugged on one end and electron-beam welded, on the other end, to a tantalum extension tube of approximately the same diameter, forming a 90-cm-long composite tube assembly. The tube assembly was suspended inside a double-walled quartz glass tube. The support flange for the tube assembly was water cooled; consequently, the exposed end of the assembly and the copper tubing connecting it to the pressure gauges and gas supply were at room temperature. Further details concerning the construction and cleaning of the heat pipes and the sample preparation are presented in Ref. 5. The specifications and sources of the metal samples used in these tests are summarized in Table I.

Table I. Test Material Specification and Source

Metal	Supplier	Purity (%)
Tin	Leytess Metal and Chemical Co., N.Y., U.S.A.	99.999
Gold	Degussa, Hanau, Germany	99.999
Silver	Leico Industries, Inc., N.Y., U.S.A.	99.999
Gallium	Leico	99.9999
Indium	Leytess	99.9999
Copper	Standard OFHC	99.95

During operation, argon, at a pressure of about 0.7 bar, fills the space surrounding the tube assembly to reduce tungsten evaporation and the mechanical load on the heat pipe wall. Cooling water flows through the annular space between the quartz tubes. The lower half of the heat pipe is heated using an R-F generator with a capacity of 30 kW. Heat is removed from the tube assembly by radiation and by convection of the surrounding argon gas. The heat input is adjusted so that the top of the isothermal zone is about 1 cm below the W-Ta joint. The outer wall temperature is measured above the heating coil, in the condenser region of the heat pipe, with an automatic precision pyrometer, corrected for the absorption in the quartz-water containment. The vapor temperature is determined by adding the calculated temperature drop through the heat pipe wall to the measured outer wall temperature. The vapor pressure is indicated on two precision pressure gauges. Provisions are made, as shown in Fig. 1, so that either helium or argon may be used as the buffer gas between the vapor and the pressure gauges.

Data taking in these tests began at a low temperature and pressure (generally at 0.1 bar) and continued, at progressively higher pressures, until either a vapor pressure of 3.4 bar was reached or the heat pipe failed. All of the heat pipes, except those containing In and Ag, failed due to penetration of the working fluid through the wall.

### 3. ACCURACY

The heat-pipe method for measuring vapor pressure has several features which enhance the accuracy in high-temperature applications compared with alternative methods [2]. Both the temperature and the pressure measurement can be made under most favorable conditions; the temperature of a large, isothermal surface and the pressure of an inert gas at room temperature are measured. Also, the vapor is produced by evaporation from the smooth surface of a thin layer of liquid without the formation of any bubbles, and therefore, the deviations from thermodynamic equilibrium are small. Finally, there is a continuous distillation of the working fluid so that, if the temperature measurements are made in the condenser area, the effect of impurities is small.

Some precautions are necessary in order to realize the full potential of the method. The inert gas should be lighter than the metal vapor, otherwise the vapor-gas interface is unstable [6] and pressure fluctuations can occur [7, 8]. Furthermore, a lighter vapor will be carried high up into the gas by convective motions and may form a solid plug in the line to the gas control system.

The measured gas pressure and tube outer surface temperature may differ from the desired vapor pressure and temperature for a number of reasons. They are as follows:

- dynamic pressure drop in the flowing vapor,
- hydrostatic pressure of the vapor and the inert gas,
- diffusion of the inert gas into the vapor,
- mixing of the inert gas and the vapor by convective instabilities,
- solution of impurities in the liquid,
- deviation of the curvature of the liquid–vapor interface from the thermodynamic equilibrium situation (due to the finite flow resistance of the capillary structure),
- incomplete thermodynamic equilibrium between vapor and liquid (due to finite evaporation and condensation rates), and
- thermal resistance of the wall of the heat pipe.

These have been discussed in detail in Ref. 9. The effects of many of these mechanisms on measurement accuracy can be minimized by careful design of the experiment. For the present experiment, errors due to the above mechanisms have been estimated to be small compared to the uncertainty in the measurement of surface temperature and gas pressure. The error due to the thermal resistance of the wall has been minimized by applying a correction for the temperature drop,  $\Delta T_w$ , between the inner and the outer walls of the heat pipe condenser. The correction is calculated from the known tube geometry, assuming the grooves to be completely filled with liquid metal, and the heat flux is estimated from the measured temperatures.

The pressure of the inert gas was measured using one of two pressure gauges with different ranges. Both were Model 61-050 Series 1500 gauges manufactured by Wallace and Tiernan–Chlorator GmbH with an accuracy of  $\pm 0.066\%$  of full scale. The estimated uncertainty in the measured vapor pressure is  $\pm 0.7$  mbar below a pressure of 1.05 bar and  $\pm 2.2$  mbar above this pressure.

The heat-pipe condenser surface temperature was measured with a Leeds and Northrup high-precision automatic optical pyrometer, Model 8642, Mark II. This pyrometer was calibrated against a standard blackbody lamp viewed through the water-filled quartz tube. The uncertainty in the absolute vapor temperature, due chiefly to uncertainty in the tube surface emissivity, is 1.1%.



#### 4. RESULTS

The final, corrected data are presented in Table II, together with the applied wall temperature drop correction,  $\Delta T_w$ . The data were fitted to the analytical form

$$\log P = A - \frac{B}{T} \quad (1)$$

with  $P$  in bar and  $T$  in K, using the least-squares method. The values for the constants  $A$  and  $B$  for each material as well as the boiling-point temperature and heat of vaporization, as calculated from the analytical expression using the Clausius–Clapeyron relation, are presented in Table III. The measured boiling-point temperature is also indicated (in parentheses) for those materials which were measured at atmospheric pressure.

#### 5. DISCUSSION

The results are summarized in Fig. 2. The scatter of the data around the regression lines, Eq. (1), is seen to be small. The present data for each element are compared, in Figs. 3 through 8, with previous measurements and estimates summarized by Nesmeyanov [10]. The present data are represented, in these plots, by the regression lines indicated by the arrows.

**Table III.** Calculated Regression Constants, Boiling Point, and Heat of Vaporization

Metal	$A$	$B$	Boiling point (K) <sup>a</sup>	Heat of vaporization (kJ·mol <sup>-1</sup> )
In	5.4731	12581	2301 (2302)	241
Ag	5.4870	13342	2434 (2427)	255
Ga	5.5174	13798	2503 (2504)	264
Cu	5.5880	15821	2834	303
Sn	5.2171	14907	2860 (2860)	285
Au	5.3997	16773	3110	321

<sup>a</sup> Numbers in parentheses refer to measured values.

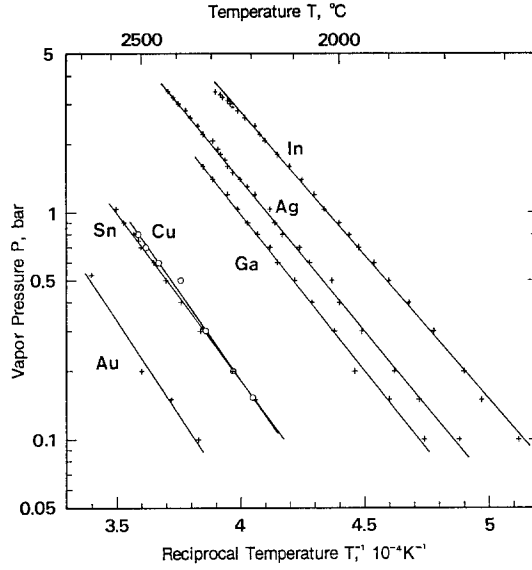


Fig. 2. Vapor pressure of In, Ag, Ga, Cu, Sn, and Au at high temperatures (heat-pipe method).

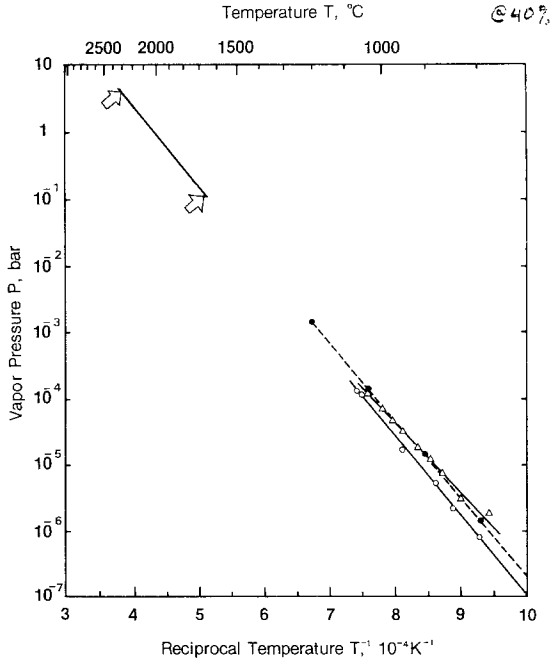


Fig. 3. Vapor pressure of indium.



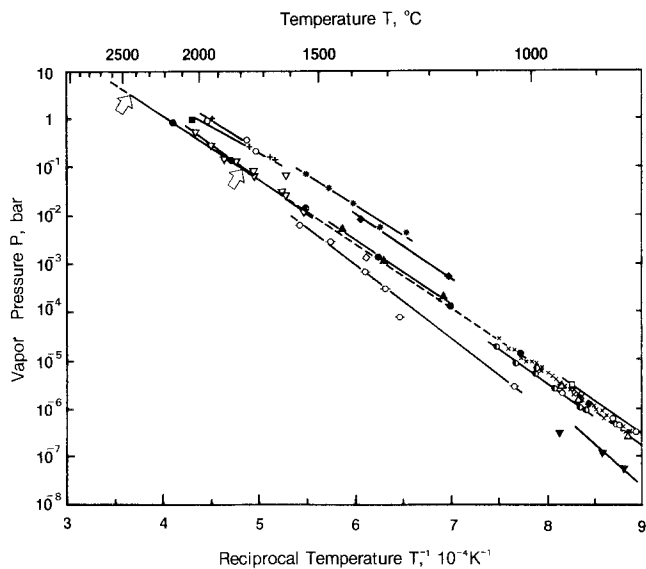


Fig. 4. Vapor pressure of silver.

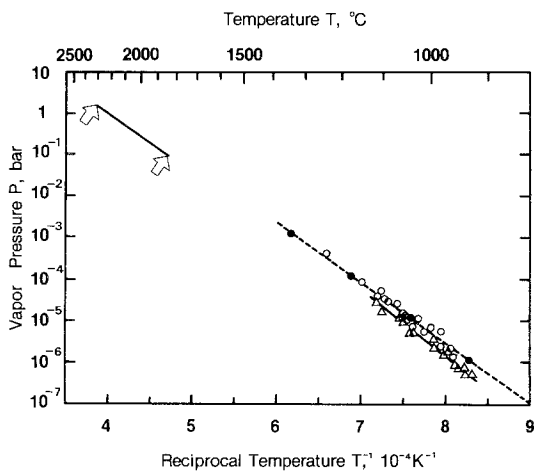


Fig. 5. Vapor pressure of gallium.

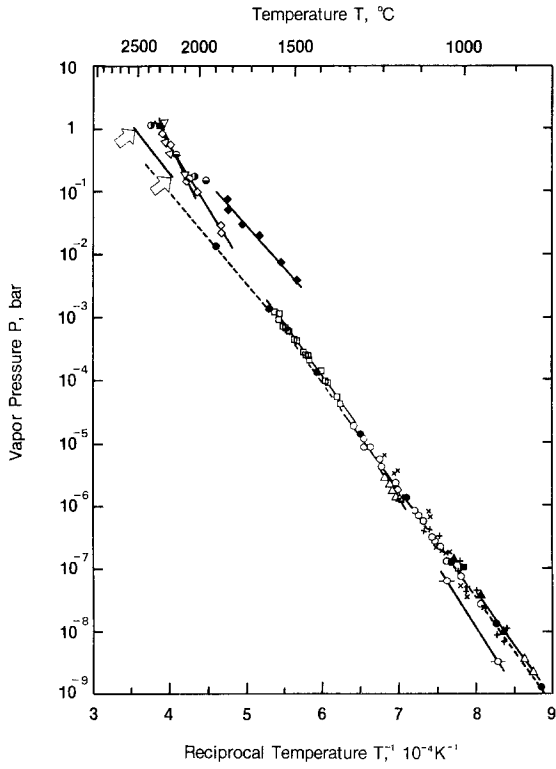


Fig. 6. Vapor pressure of copper.

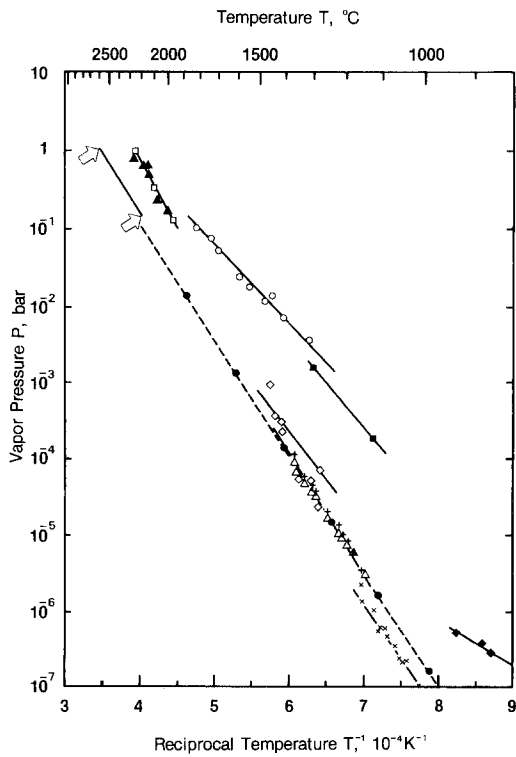


Fig. 7. Vapor pressure of tin.

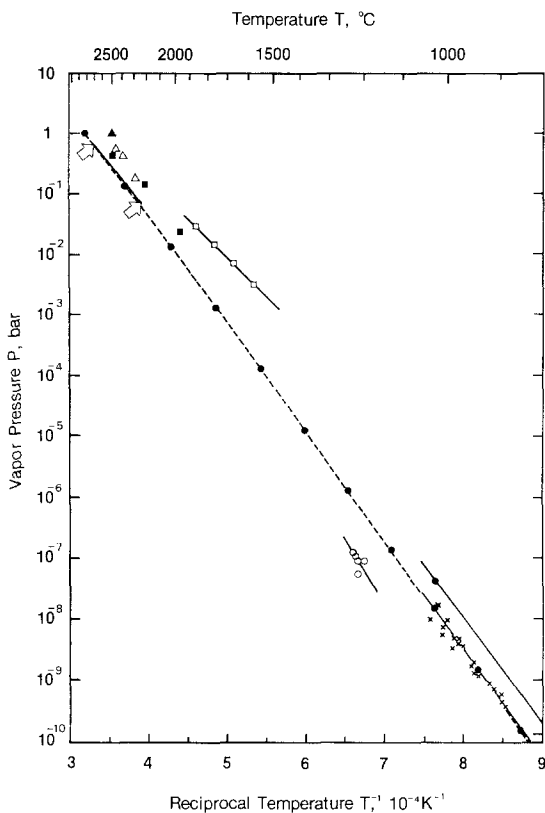


Fig. 8. Vapor pressure of gold.

In the cases of In and Ga (Figs. 3 and 5), no data or estimates are presented in Ref. 10 in the temperature range investigated here. The results for Ag (Fig. 4) agree remarkably well with the estimate of Nesmeyanov. The present results for Au (Fig. 8) are also in good agreement with his estimate. The data for Sn (Fig. 7) fall just beyond the range of the estimate provided in Ref. 10 but are consistent with the low-temperature data on which the estimate was based. The largest deviation between the present results and the estimate of Nesmeyanov is for Cu (Fig. 6). Here again, the estimate was based on an extrapolation of low-temperature data.

The boiling points of these metals have also been estimated by Hultgreen et al. [11] from data taken at lower temperature. These boiling points are as follows: In, 2346 K; Ag, 2436 K; Ga, 2478 K; Cu, 2836 K; Sn, 2876 K; and Au, 3130 K. These values agree well with the data presented here.

## ACKNOWLEDGMENT

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